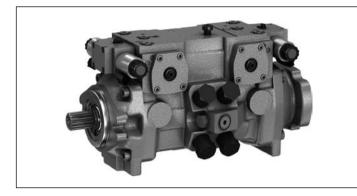


Axial piston variable double pump A22VG series 40

RE 93221

Edition: 05.2014 Replaces: 06.2012



Features

- Variable double pump with two axial piston rotary groups of swashplate design for hydrostatic drives in a closed circuit
- The flow is proportional to the drive speed and displacement.
- The flow can be infinitely varied by adjusting the swashplate angle.
- Flow direction changes smoothly when the swashplate is moved through the neutral position.
- Only one shared port for case drain fluid for both circuits
- Compact design for tight installation conditions

Note

Only for series no smaller than 200 units per year. Please contact us regarding smaller series.

- Size 45
- Nominal pressure 380 bar
- Maximum pressure 420 bar
- Closed circuit

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2 **A22VG series 40** | Axial piston variable double pump Ordering code

Ordering code

| | 01 | 02 | 03 | 04 | 05 | 06 | 07 | 08 | 09 | | | 10 | 11 | 12 | 13 | 14 | 15 | 16 | 17 | 18 | 19 | | 20 |
|-------|--|----------|----------|--------------------|----------|---------|----------|---------|---------|--------|------|--------|---------|----------|--------|----------|------------|--------|-------|----|----|-----------|------------|
| A | 22V | G | 045 | | | | | | | / | | 40 | Α | | Ν | B2 | S 7 | 3 | | | Α | - | |
| \xial | piston | unit | | | | | | | | | | | | | | | | | | | | | |
| 01 | Swash | | design | , varia | able, no | ominal | press | ure 38 | 0 bar, | max | imu | im pre | essure | e 420 k | bar | | | | | | | | A22\ |
| Oper | ating m | node | | | | | | | | | | | | | | | | | | | | | - |
| 02 | Doubl | | p, clos | ed cir | cuit | | | | | | | | | | | | | | | | | | G |
| Sizo | (NG) | - | | | | | | | | | | | | | | | | | | | | | - <u>I</u> |
| 03 | Geom | etric c | lisplac | ement | . see t | echnic | al data | a on pa | age 7 | | | | | | | | | | | | | 045 | 1 |
| | rol devi | | | | , | | | | | | | | | | | | | | | | | | J |
| 04 | Propo | | l contr | ol hvd | raulic | | | | | | | | | witho | ut nei | itral po | osition | switch | ו | | | | HW2 |
| 01 | mecha | | | | | ft with | n lever, | free p | ositio | n1) | | | | | | l posit | | | | | | | нжа |
| | Propo | rtiona | l contr | ol elec | ctric | | | | | | | | | | 2 V DC | | | | | | | | EP1 |
| | | | | | | | | | | | | | | U = 24 | 4 V DC | ; | | | | | | | EP2 |
| | Hydra | ulic co | ontrol, | direct | opera | ted | | | | | | | | | | | | | | | | | HT1 |
| | Electri | ic con | trol, di | rect o | perate | d; | | | | | | | | U = 1 | 2 V DC | ; | | | | | | | ET1 |
| | two pressure reducing valves per circuit U = 24 V DC | | | | | | | | ET2 | | | | | | | | | | | | | | |
| Conn | ector f | or sol | enoids | ²⁾ (see | e page | 23) | | | | | | | | | | | | | | | | | - |
| 05 | Witho | | | | | | , only t | for hyc | Iraulic | cont | rol |) | | | | | | | | | | | 0 |
| | DEUTS | SCH – | molde | d con | nector | , 2-pin | – with | nout su | ppres | sor c | lioc | le | | | | | | | | | | | Р |
| Swiv | el angle | sens | or (see | nage | 22) | | | | | | | | | | | | | | | | | | 4 |
| 06 | Witho | | | | | | | | | | | | | | | | | | | | | | 0 |
| | Electri | | | | | ounted | 3) | | | | | | | | | | | | | | | | R |
| | | | | | | | | | | | | | | | | | | нพ | нт | E | | ст | <u> </u> |
| 07 | pressu Ports | - | | | | | | | | | | | | | | | 1 | • | - I | | | <u>ЕТ</u> | 1 |
| 01 | Ports | | | | | | | | | | | | | | | | | - | • | | | - | 3 |
| | Ports | | | X | | | | | | | | | | | | | | • | - | • | | • | 4 |
| | Ports | | | ,, , , 4 | | | | | | | | | | | | | | - | • | - | | - | 5 |
| Mack | anical | | | | | 22) | | | | | | | | | | | I | | | | | | <u> </u> |
| 08 | Witho | | | | | | | | | | | | | | | | | | | | | | 0 |
| 00 | | | nechan | | | | extern | allv ad | liustat | ole. o | n o | ppos | ite sic | le to se | ervice | line po | orts | | | | | | F |
| | | | | | | | | | | , . | | ppoo | | | | | | 1114/ | | | | FT | <u> </u> |
| | ontrol v | | | | | | | | | | | | | | | | | HW | нт | El | | ET | 0 |
| 09 | Witho | | alve fix | | | | | | | | | | | | | | | • | • | • | | • | 1 |
| | | | | keu se | tting | | | | | | | | | | | | | • | • | - | | | <u> </u> |
| Serie | r | . 1 : | | | | | | | | | | | | | | | | | | | | | 40 |
| 10 | Series | 5 4, INC | | | | | | | | | | | | | | | | | | | | | 40 |
| | | | | | | | | | | | | | | | | | | | | | | | |
| | i guratio ANSI, | | | | | | | | | | | | | | | | | | | | | | A |

 On delivery, the position of the lever may differ from that shown in the brochure or drawing. If necessary, the position of the lever can be adjusted by the customer.

2) Connectors for other electric components can deviate.

3) Please contact us if the swivel angle sensor is used for control

| (| 01 | 02 | 03 | 04 | 05 | 06 | 07 | 08 | 09 | | | 10 | 11 | 12 | 13 | 14 | 15 | 16 | 17 | 18 | 19 | | 20 |
|-------|-----------|--------------|----------|-----------|-----------------|----------|---------|--------|--------|------|--------|--------------------|--------|--------|--------|----|--------|--------|------|----|----|---|------|
| A | 22V | G | 045 | | | | | | | | / | 40 | Α | | Ν | B2 | S7 | 3 | | | Α | - | |
| Direc | tion of | rotat | ion | | | | | | | | | | | | | | | | | | | | |
| 12 | Viewe | d on c | drive sh | naft | | | | | | | | | | | | С | lockwi | se | | | | | R |
| | | | | | | | | | | | | | | | | с | ounter | -clock | wise | | | | L |
| Seali | ng mat | erial | | | | | | | | | | | | | | | | | | | | | |
| 13 | NBR (| nitrile | -rubbe | r), shaf | t sea | l in FKI | M (fluc | roelas | tome | r) | | | | | | | | | | | | | Ν |
| Mour | nting fla | ange | | | | | | | | | | | | | | | | | | | | | |
| 14 | SAE J | 744, 1 | .01-2 | | | | | | | | | | | | | | | | | | | | B2 |
| Drive | shaft | (perm | issible | input to | orque | e, see p | bage 1 | 7) | | | | | | | | | | | | | | | |
| 15 | Spline | ed sha | ft ANS | B92.1a | a, 1 1 | /4 in 1 | 4T 12, | /24DP | | | | | | | | | | | | | | | S7 |
| Servi | ce line | ports | ; | | | | | | | | | | | | | | | | | | | | |
| 16 | 1 | - | | and B, I | eft (\ | viewed | on dri | ve sha | ft) | | | | | | | | | | | | | | 3 |
| Boos | t pump | 4) | | | | | | | | | | | | | | | | | | | | | |
| 17 | <u> </u> | | ost pun | np (star | ndarc | 4) | | | | | | | | | | | | | | | | | U |
| | Boost | pump |) | | | | | | | | | | | | | | | | | | | | F |
| Throu | ugh driv | ve (m | ounting | g optior | ıs, se | e page | 19) | | | | | | | | | | | | | | | | |
| 18 | Flange | | | | - | | | Hub | for sp | olin | ed sl | haft ⁵⁾ | | | | | | | | | | | |
| | Diame | eter | М | ounting | ⁶⁾ [| Designa | ation | Diam | eter | | | | | Desig | nation | | | | | | | | |
| | 101-2 | (B) | ~ |) | E | B2 | | 7/8 i | n : | 13T | Г 16/ | 32DP | | S4 | | | | | | | | | B2S4 |
| | | | | | | | | 1 in | | 15T | Г 16/ | 32DP | | S5 | | | | | | | | | B2S5 |
| Press | sure-rel | lief va | lve | | | | | | | | | | | | | | | | | | | | |
| 19 | High-p | oressu | re relie | ef valve, | dire | ct opei | rated, | withou | ıt byp | ass | s (foi | r value | s, see | page 2 | 20) | | | | | | | | Α |
| Stand | dard / s | specia | l versi | on | | | | | | | | | | | | | | | | | | | |
| 20 | Stand | | | | | | | | | | | | | | | | | | | | | | 0 |
| | Stand | ard ve | ersion v | vith ins | tallat | ion var | iants, | e.g. T | ports | ag | ainst | t stanc | lard o | oen or | closed | k | | | | | | | Y |
| | Specia | al vers | sion | | | | | | | | | | | | | | | | | | | | S |

• = Available - = Not available

Notes

- Observe the project planning notes on page 26!
- A pressure cut-off is not available for this unit.
- Preservation:
 - up to 12 months as standard
 - up to 24 months long-term

(state in plain text when ordering)

⁴⁾ Pressure or suction filtration required. To be supplied by customer. Boost pressure inlet at port G, a DA control valve is used at port G1.

⁵⁾ Hub for splined shaft according to ANSI B92.1a

⁶⁾ Mounting drillings pattern viewed on through drive with control at top

4 **A22VG series 40** | Axial piston variable double pump Hydraulic fluids

Hydraulic fluids

The A22VG variable double pump is designed for operation with HLP mineral oil according to DIN 51524.

Application notes and requirements for hydraulic fluids should be taken from the following data sheets before the start of project planning:

- 90220: Hydraulic fluids based on mineral oils and related hydrocarbons
- 90221: Environmentally acceptable hydraulic fluids
- 90222: Fire-resistant, water-free hydraulic fluids (HFDR/HFDU)

Details regarding the selection of hydraulic fluid

The hydraulic fluid should be selected such that the operating viscosity in the operating temperature range is within the optimum range (v_{opt} see selection diagram).

Note

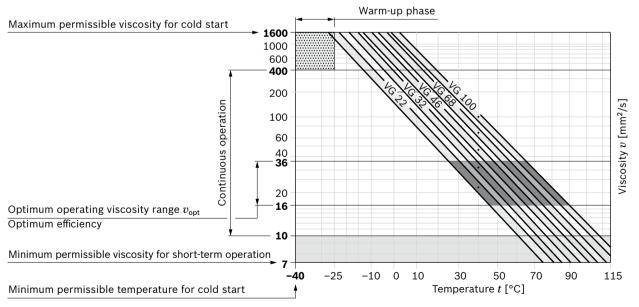
At no point of the component may the temperature be higher than 115 °C. The temperature difference specified in the table is to be taken into account when determining the viscosity in the bearing.

If the above conditions cannot be maintained due to extreme operating parameters, please contact the responsible member of staff at Bosch Rexroth.

Viscosity and temperature of hydraulic fluids

| | Viscosity | Temperature | Comment |
|----------------------|---|---|---|
| Cold start | $v_{\rm max} \le 1600 \ {\rm mm^2/s}$ | $\theta_{\rm St} \ge -40 \ ^{\circ}{\rm C}$ | $t \le 3 \text{ min}, n \le 1000 \text{ rpm}$, without load $p \le 50 \text{ bar}$ |
| Permissible temper | Permissible temperature difference | | between axial piston unit and hydraulic fluid in the system |
| Warm-up phase | v < 1600 to 400 mm ² /s | θ = -40 °C to -25 °C | At $p \le 0.7 \times p_{\text{nom}}$, $n \le 0.5 \times n_{\text{nom}}$ and $t \le 15$ min |
| Continuous operation | v = 400 to 10 mm ² /s | | This corresponds, for example on the VG 46, to a temperature range of +5 °C to +85 °C (see selection diagram) |
| | | <i>θ</i> = -25 °C to +110 °C | measured at port T Note the permissible temperature range of the shaft seal (ΔT = approx. 5 K between the bearing/shaft seal and port T) |
| | v_{opt} = 36 to 16 mm ² /s | | Range of optimum operating viscosity and efficiency |
| Short-term operation | $v_{min} \ge 7 \text{ mm}^2/\text{s}$ | | $t < 3 \min, p < 0.3 \times p_{nom}$ |

Selection diagram



Filtration of the hydraulic fluid

Finer filtration improves the cleanliness level of the hydraulic fluid, which increases the service life of the axial piston unit.

A cleanliness level of at least 20/18/15 is to be maintained according to ISO 4406.

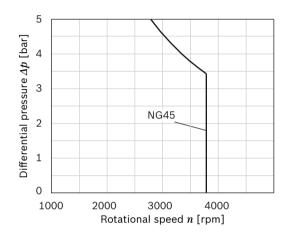
We recommend, depending on the system and application, for the A22VG: filter cartridges $\beta_{20} \ge 100$.

At very high hydraulic fluid temperatures (90 °C to maximum 110 °C, measured at port T), a cleanliness level of at least 19/17/14 according to ISO 4406 is necessary.

Shaft seal

Permissible pressure loading

The service life of the shaft seal is influenced by the speed of the axial piston unit and the leakage pressure in the housing (case pressure). Momentary pressure spikes (t < 0.1 s) of up to 10 bar are permitted. The service life of the shaft seal decreases with increasing frequency of pressure spikes and increasing mean differential pressure. The case pressure must be equal to or higher than the ambient pressure.

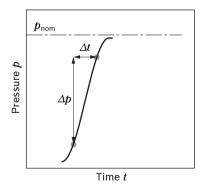


The FKM shaft seal may be used for leakage temperatures from -25 °C to +115 °C. For application cases below -25 °C, an NBR shaft seal is required (permissible temperature range: -40 °C to +90 °C).

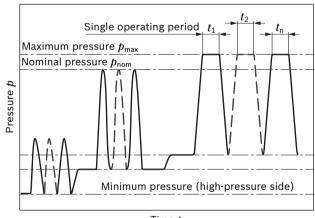
Operating pressure range

| Pressure at service line port A or B | | Definition |
|--|-------------------------------|--|
| Nominal pressure $p_{\sf nom}$ | 380 bar absolute | The nominal pressure corresponds to the maximum design pressure. |
| Maximum pressure p_{\max} | 420 bar absolute | The maximum pressure corresponds to the maximum operating pressure within |
| Single operating period | 10 s | the single operating period. The sum of the single operating periods must not exceed |
| Total operating period | 300 h | the total operating period. |
| Minimum pressure (high-pressure side) | 25 bar absolute | Minimum pressure at the high-pressure side (A or B) which is required in order to prevent damage to the axial piston unit. |
| Minimum pressure (low-pressure side) | 10 bar above case pressure | Minimum pressure at the low-pressure side (A or B) which is required in order to prevent damage to the axial piston unit. Boost pressure setting must be higher depending on system. |
| Rate of pressure change $R_{A max}$ | 9000 bar/s | Maximum permissible rate of pressure build-up and reduction during a pressure change over the entire pressure range. |
| Boost pump | | |
| Nominal pressure p_{Spnom} | 25 bar absolute | |
| Maximum pressure $p_{Sp max}$ | 30 bar absolute | |
| Pressure at suction port S (inlet) | | |
| Continuous $p_{\text{S min}}$ ($v \le 30 \text{ mm}^2/\text{s}$) | ≥0.8 bar absolute | |
| Short-term, on cold start (t < 3 min) | ≥0.5 bar absolute | |
| Maximum pressure $p_{ m Smax}$ | ≤5 bar absolute | |
| Control pressure | | |
| Minimum control pressure $p_{ m St\ min}$ | | To ensure the function of the control, a minimum control pressure $p_{ m Stmin}$ at |
| Controls EP and HW | 18 bar above case pressure | n = 2000 rpm is required depending on the rotational speed and operating pressure |
| Controls ET and HT | 25 bar above case pressure | |

▼ Rate of pressure change R_{A max}



Pressure definition



Time t

Total operating period = $t_1 + t_2 + ... + t_n$

Note

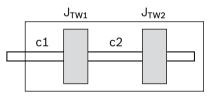
Operating pressure range valid when using hydraulic fluids based on mineral oils. Values for other hydraulic fluids, please contact us.

| Size | | | NG | | 45 |
|--------------------------------|--|----------------------|-----------------------|--------------------|--------------------|
| Displacement geometric, | variable pump (for ea | ach rotary group) | $V_{\sf gmax}$ | cm ³ | 2 x 46 |
| per revolution | boost pump (at p = 2 | 5 bar) | V_{gSp} | cm ³ | 14.9 |
| Rotational speed ¹⁾ | maximum at $V_{g max}$ | | $n_{\sf nom}$ | rpm | 3300 ⁶⁾ |
| | limited maximum ²⁾ | | $n_{\max 1}$ | rpm | 3550 |
| | intermittent maximu | m ³⁾ | $n_{\max 2}$ | rpm | 3800 |
| | minimum | | n_{\min} | rpm | 500 |
| Flow | at $V_{ m gmax}$ and $n_{ m nom}$ | | $q_{ m v}$ | l/min | 2 x 152 |
| Power ⁴⁾ | at $V_{ m gmax}$, $n_{ m nom}$ and ${\it \Delta p}$ | e = 380 bar | Р | kW | 192 |
| Torque ⁴⁾ | at $V_{g max}$ and | ∆p = 300 bar | Т | Nm | 556 |
| | | Δp = 100 bar | Т | Nm | 146 |
| Rotary stiffness drive shaft | 1 1/4 in S7 | Pump 1 | <i>c</i> ₁ | Nm/rad | 73804 |
| | | Pump 2 | <i>c</i> ₂ | Nm/rad | 23066 |
| Moment of inertia | rotary group 1 | | J _{TW1} | kgm ² | 0.003327 |
| (see graphic below) | rotary group 2 | | J_{TW2} | kgm² | 0.003293 |
| Maximum angular acceleratio | on for each rotary group | 5) | α | rad/s ² | 4000 |
| Case volume | | | V | L | 1.7 |
| Weight with HT control (appr | ox.) | | m | kg | 53 |

Technical data

| Flow $q_v = \frac{V_g \times n \times \eta_v}{1000}$ [1 | [l/min] |
|---|---------|
| Torque $T = \frac{V_{g} \times \Delta p}{20 \times \pi \times \eta_{mh}}$ [1] | [Nm] |
| Power $P = \frac{2 \pi \times T \times n}{60000} = \frac{q_v \times \Delta p}{600 \times \eta_t} [$ | [kW] |
| Кеу | |
| V _g = Displacement per revolution [cm ³] | |
| Δp = Differential pressure [bar] | |
| <i>n</i> = Rotational speed [rpm] | |
| η_v = Volumetric efficiency | |
| η_{mh} = Mechanical-hydraulic efficiency | |
| $\eta_{\rm t}$ = Total efficiency ($\eta_{\rm t} = \eta_{\rm v} \times \eta_{\rm mh}$) | |

Spring-mass system with moment of inertia



Notes

- Theoretical values, without efficiency levels and tolerances; values rounded
- Operation above the maximum values or below the minimum values may result in a loss of function, a reduced service life or in the destruction of the axial piston unit. Bosch Rexroth recommends checking the loading by means of testing or calculation / simulation and comparison with the permissible values.
- Transport and storage
 - $\theta_{min} \ge -50 \ ^{\circ}C$
 - θ_{opt} = +5 °C to +20 °C

1) The values are valid:

- for the optimum viscosity range of v_{opt} = 36 to 16 mm²/s with hydraulic fluid based on mineral oil
- 2) limited maximum speed:
- At half corner power (e.g., at $V_{g max}$ and p_{nom} /2) 3) Intermittent maximum speed at:
 - high idle
 - overspeed: Δp = 70 to 150 bar and V_{g max}
 - reversing peaks: Δp < 300 bar and t < 0.1 s.

4) Without boost pump

⁵⁾ The data are valid for values between the minimum required and maximum permissible rotational speed. Valid for external excitation (e. g. diesel engine 2 to 8 times rotary frequency; cardan shaft twice the rotary frequency). The limit value applies for a single pump only. The load capacity of the connection parts must be considered.

⁶⁾ When using a boost pump, please consult with the responsible plant.

8 **A22VG series 40** | Axial piston variable double pump Technical data

Permissible radial and axial forces of the drive shaft

| Size | | NG | | 45 |
|------------------------------------|---------------------|------------------|----|-------|
| Drive shaft | | | in | 1 1/4 |
| Maximum radial force at distance a | ↓F _{q ⊓} | $F_{q \max}$ | Ν | 3190 |
| (from shaft collar) | | a | mm | 24 |
| Maximum axial force | F _{ax} ±≠€ | $\pm F_{ax max}$ | N | 1500 |

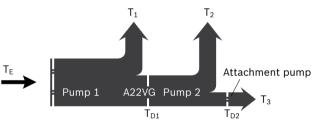
Note

Special requirements apply in the case of belt drive and cardan shaft. Please contact us.

Permissible input and through-drive torques

| Size | | | NG | | 45 |
|---|---|----------|---------------------|----|---------------------------|
| Torque at V _{g max} and ${\it \Delta p}$ | Torque at V _{g max} and Δp = 380 bar ¹⁾ | | | | 556 |
| Maximum input torque | at drive shaft ²⁾ | | | | |
| | S7 | 1 1/4 in | $T_{E\ max}$ | Nm | 602 |
| Maximum through-drive torque | | | T _{D1 max} | Nm | 300 |
| | | | T _{D2 max} | Nm | $T_{D2 perm} = 300 - T_2$ |

▼ Torque distribution



| Torque – A22VG | 1st pump | T_1 | | |
|-----------------------|----------|-----------------|---|-------------------|
| | 2nd pump | T_2 | | |
| Torque – attachment p | ump | T_3 | | |
| Input torque | | T_{E} | = | $T_1 + T_2 + T_3$ |
| | | T_{E} | < | $T_{E\ max}$ |
| Through-drive torque | | T_{D1} | | |
| | | T_{D2} | | |

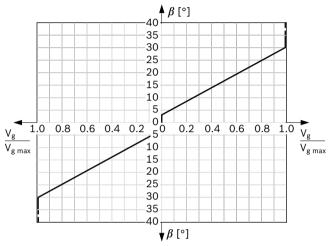
1) Efficiency not considered

2) For drive shafts without radial force

HW - proportional control hydraulic, mechanical servo

The output flow of the pump is infinitely variable between 0 to 100%, proportional to the swivel angle of the control lever.

A feedback lever, connected to the stroking piston maintains the pump flow for a given position of the control lever. If the pump is also equipped with a DA control valve (see page 15), automotive operation is possible for travel drives.



Swivel angle β at the control lever for pump displacement change:

- Start of control at β = ±3°
- End of control at β (max. displacement $V_{g max}$) at ±30°
- Rotation limiting β of the control lever (internal) ±38°

The maximum required torque at the lever is 170 Ncm. To prevent damage to the HW control module, a mechanical stop must be provided by the customer for the HW control lever.

Note

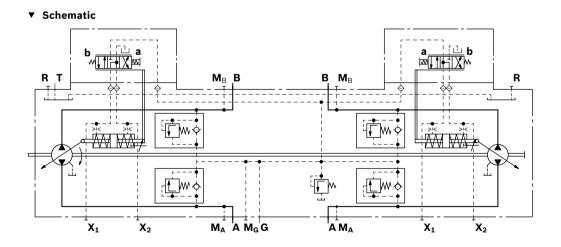
Spring centering enables the pump, depending on pressure and rotational speed, to move automatically to the neutral position (V_g = 0) as soon as there is no longer any torque on the control lever of the HW control module (regardless of deflection angle).

Variation: Neutral position switch

The switch contact in the neutral position switch is closed when the control lever on the HW control module is in its neutral position. The switch opens when the control lever is moved out of neutral in either direction. Thus, the neutral position switch provides a monitoring

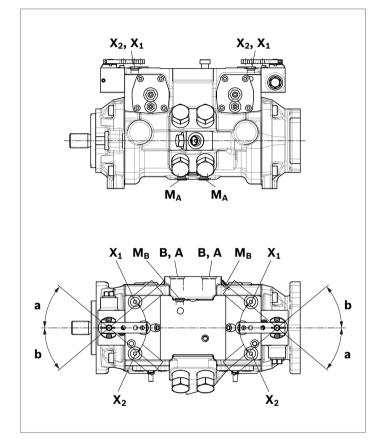
function for drive units that require the pump to be in the neutral position during certain operating conditions (e.g. starting diesel engines).

| Technical data | |
|--------------------|---|
| Load capacity | 20 A (continuous), without switching operations |
| Switching capacity | 15 A / 32 V (resistive load) |
| | 4 A / 32 V (inductive load) |
| Connector version | DEUTSCH DT04-2P-EP04 |
| | (Mating connector, see page 23) |



| 10 | A22VG series 40 Axial piston variable double pump |
|----|---|
| | HW – proportional control hydraulic, mechanical servo |

| Assignment of direction of rotation, control and flow direction | | | | | | | | |
|---|----------------|----------------|----------------|----------------|----------------|-------------------|----------------|----------------|
| Direction of rotation | clockwise | clockwise c | | | counter-c | counter-clockwise | | |
| Pump | Pump 1 | | Pump 2 | | Pump 1 | | Pump 2 | |
| Lever direction | a | b | а | b | a | b | a | b |
| Control pressure (X ₃ , X ₄ | X ₂ | X ₁ | X1 | X ₂ | X ₂ | X ₁ | X1 | X ₂ |
| optional, see page 21) | X ₄ | X ₃ | X ₃ | X ₄ | X4 | X ₃ | X ₃ | X ₄ |
| Flow direction | A to B | B to A | B to A | A to B | B to A | A to B | A to B | B to A |
| Operating pressure | M _B | M _A | M _A | M _B | M _A | M _B | M _B | M _A |

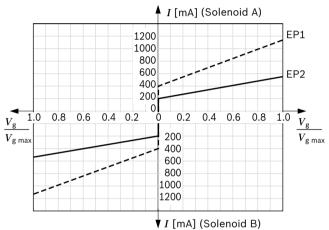


EP – Proportional control electric

The output flow of the pump is infinitely variable between 0 to 100%, proportional to the electrical current supplied to solenoid **a** or **b**.

The electrical energy is converted into a force acting on the control spool.

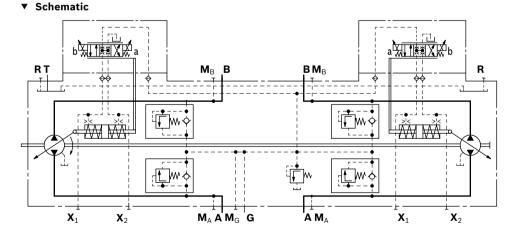
This valve spool then directs control oil into and out of the stroking cylinder to adjust pump displacement as required. A feedback lever, connected to the stroking piston maintains the pump flow for a given current within the control range. If the pump is also equipped with a DA control valve (see page 15), automotive operation is possible for travel drives.



| Technical data, solenoid | EP1 | EP2 | | | | |
|--------------------------------------|--|-------------|--|--|--|--|
| Voltage | 12 V (±20%) | 24 V (±20%) | | | | |
| Control current | | | | | | |
| Beginning of control at V_{g} = 0 | 400 mA | 200 mA | | | | |
| End of control at $V_{g max}$ | 1115 mA | 560 mA | | | | |
| Current limit | 1.54 A | 0.77 A | | | | |
| Nominal resistance (at 20 °C) | 5.5 Ω | 22.7 Ω | | | | |
| Dither frequency | 100 Hz | 100 Hz | | | | |
| Duty cycle | 100 % | 100 % | | | | |
| Type of protection, see connector ve | Type of protection, see connector version on page 23 | | | | | |

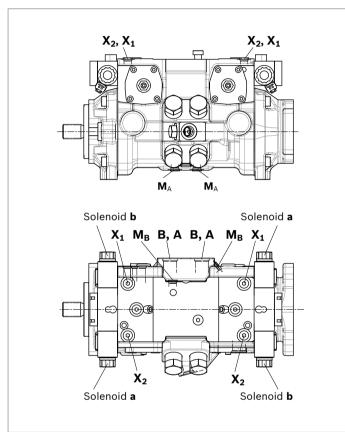
Various BODAS controllers with application software and amplifiers are available for controlling the proportional solenoids.

Further information can also be found on the internet at www.boschrexroth.com/mobile-electronics.



A22VG series 40 | Axial piston variable double pump EP – Proportional control electric 12

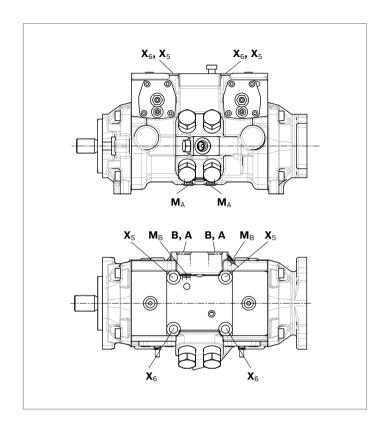
| Assignment of direction of | rotation, cont | rol and flow d | lirection | | | | | |
|----------------------------------|----------------|----------------|----------------|----------------|----------------|-------------------|----------------|----------------|
| Direction of rotation | clockwise | clockwise c | | | counter-c | counter-clockwise | | |
| Pump | Pump 1 | | Pump 2 | | Pump 1 | | Pump 2 | |
| Actuation of solenoid | а | b | а | b | a | b | а | b |
| Control pressure (X_3 , X_4 | X ₂ | X1 | X1 | X ₂ | X ₂ | X1 | X1 | X ₂ |
| optional, see page 21) | X ₄ | X ₃ | X ₃ | X ₄ | X ₄ | X ₃ | X ₃ | X ₄ |
| Flow direction | A to B | B to A | B to A | A to B | B to A | A to B | A to B | B to A |
| Operating pressure | M _B | M _A | M _A | MB | M _A | M _B | M _B | M _A |



HT - Hydraulic control, direct operated

With the direct hydraulic control, the flow of the pump is influenced by a hydraulic control pressure that is applied directly to the stroking piston through X_5 or X_6 . Flow direction is determined by which control pressure port is pressurized (refer to table below). Pump displacement is infinitely variable and proportional to the applied control pressure, but is also influenced by system pressure and pump drive speed. Maximum permissible control pressure: 30 bar Use of the HT control requires a review of the engine and vehicle parameters to ensure that the pump is set up correctly. We recommend that all HT applications be reviewed by a Bosch Rexroth application engineer. The DA control valve only becomes effective if the pilot control device used for controlling the HT control is supplied from port **Y**.

Schematic



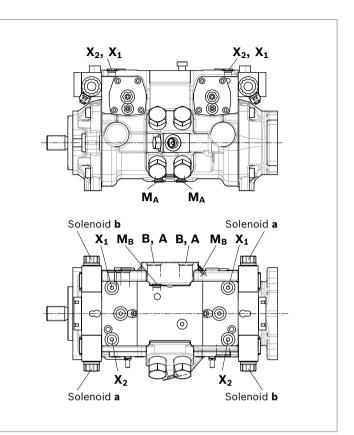
RТ \mathbf{M}_{B} \mathbf{M}_{B} R R R ₩¢ ŢŴŶ Ht M WÓ ć **X**5 **X**5 \mathbf{X}_{6} **M**_A **A M**_G **G** \mathbf{X}_{6} A MA

| Assignment of direction of rotation, control and flow direction | | | | | | | | | |
|---|----------------|----------------|----------------|----------------|----------------|-------------------|----------------|----------------|--|
| Direction of rotation | clockwise | clockwise | | | | counter-clockwise | | | |
| Pump | Pump 1 | | Pump 2 | | Pump 1 | | Pump 2 | | |
| Control pressure (X ₃ , X ₄ | X ₆ | X5 | X ₅ | X ₆ | X ₆ | X ₅ | X ₅ | X ₆ | |
| optional, see page 21) | X ₄ | X ₃ | X ₃ | X ₄ | X ₄ | X ₃ | X ₃ | X4 | |
| Flow direction | A to B | B to A | B to A | A to B | B to A | A to B | A to B | B to A | |
| Operating pressure | M _B | M _A | M _A | M _B | M _A | M _B | M _B | M _A | |

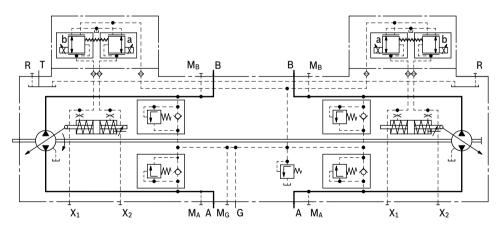
ET – Electric control, direct operated

The output flow of the pump is infinitely variable in the range 0 to 100%. Depending on the preselected current I (mA) at solenoids **a** and **b** of the pressure reducing valves, the stroking cylinder of the pump is proportionally supplied with control pressure. The pump displacement that arises at a certain control current is dependent on the rotational speed and operating pressure of the pump. A different flow direction is associated with each pressure reducing valve. Maximum permissible control pressure: 30 bar

| Technical data, solenoid | ET1 | ET2 | | | |
|--|-------------|-------------|--|--|--|
| Voltage | 12 V (±20%) | 24 V (±20%) | | | |
| Current limit | 1.54 A | 0.77 A | | | |
| Nominal resistance (at 20 °C) | 5.5 Ω | 22.7 Ω | | | |
| Dither frequency | 100 Hz | 100 Hz | | | |
| Duty cycle 100 % 100 % | | | | | |
| Type of protection, see connector version on page 23 | | | | | |



Schematic



Assignment of direction of rotation, control and flow direction

| Assignment of uncetion of to | cation, conti | | needon | | | | | |
|---|----------------|----------------|----------------|----------------|----------------|-------------------|----------------|----------------|
| Direction of rotation | clockwise | clockwise | | | | counter-clockwise | | |
| Pump | Pump 1 | | Pump 2 | | Pump 1 | | Pump 2 | |
| Actuation of solenoid | a | b | а | b | a | b | а | b |
| Control pressure (X ₃ , X ₄ op- | X ₂ | X ₁ | X ₁ | X ₂ | X ₂ | X1 | X1 | X ₂ |
| tional, see page 21) | X ₄ | X ₃ | X ₃ | X ₄ | X ₄ | X ₃ | X ₃ | X ₄ |
| Flow direction | A to B | B to A | B to A | A to B | B to A | A to B | A to B | B to A |
| Operating pressure | M _B | M _A | M _A | M _B | M _A | M _B | M _B | M _A |

DA - Control valve, fixed setting

Speed related pilot pressure supply

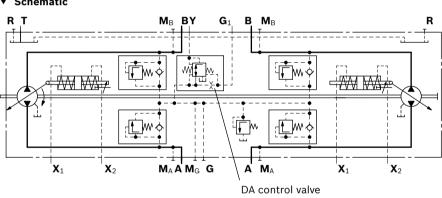
The DA closed loop control is an engine speed-dependent system for travel drives. The built-in DA control valve generates a pilot pressure which is proportional to pump (engine) drive speed. The pump displacement is infinitely variable in each flow direction and is influenced by both pump drive speed and system pressure.

Increasing the pump drive speed causes the DA control valve to generate a higher pilot pressure with a resulting increase in the flow from the pump.

Depending on the selected pump operating characteristics, increasing system pressure (e.g. machine load) causes the pump to swivel back towards a smaller displacement. Diesel engine overload protection (anti-stall) is achieved by the combination of this pressure-related pump de-stroking, and the reduction of pilot pressure as the engine speed drops. Any additional power requirement, e.g. for hydraulic functions from attachments, could cause the engine speed to drop further. This would cause a further reduction in pilot pressure and thus of pump displacement. Automatic power distribution and full exploitation of the available power are achieved in this way, both for the travel drive and for the implement hydraulics, with priority given to the implement hydraulics.

The DA control valve can also be used in pumps with EP, HT and HW control modules to protect the combustion engine against overload.

DA closed loop control is only suitable for certain types of drive systems and requires review of the engine and vehicle parameters to ensure that the pump is used correctly and that machine operation is safe and efficient. We recommend that all DA applications be reviewed by a Bosch Rexroth application engineer.

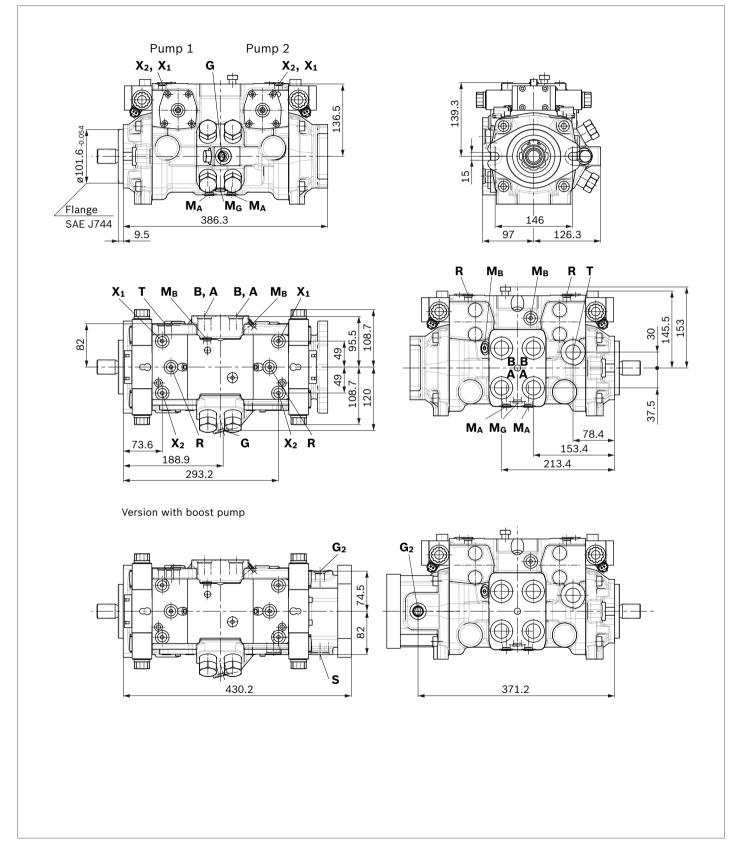


▼ Schematic

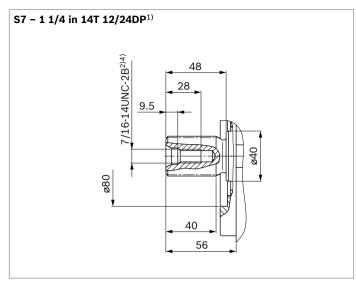
Dimensions size 45

EP – Proportional control electric

ET – Electric control, direct operated



▼ Splined shaft SAE J744



| Ports | | Standard ³⁾ | Size [in] ⁴⁾ | p _{max abs} [bar] ⁵⁾ | State ⁷⁾ |
|---|--|------------------------|--------------------------|--|---------------------|
| А, В | Working port | ISO 11926 | 1 1/16-12 UN-2B; 20 deep | 420 | 0 |
| S | Suction port (only for boost pump) | ISO 11926 | 1 5/16-12 UN-2B; 20 deep | 5 | 0 |
| т | Drain port | ISO 11926 | 1 1/16-12 UN-2B; 20 deep | 3 | 0 |
| R | Air bleed | ISO 11926 | 9/16-18 UNF-2B; 13 deep | 3 | Х |
| X ₁ , X ₂ | Control pressure (upstream of orifice, only HP, HW, EP, ET) | ISO 11926 | 9/16-18 UNF-2B; 13 deep | 30 | Х |
| X ₅ , X ₆ | Control pressure (upstream of orifice, HT only) | ISO 11926 | 9/16-18 UNF-2B; 13 deep | 30 | 0 |
| X ₃ , X ₄ ⁶⁾ | Stroking chamber pressure | ISO 11926 | 7/16-20 UNF-2B; 12 deep | 30 | Х |
| Y | Pilot pressure, outlet (only for DA control valve) | ISO 11926 | 9/16-18 UNF-2B; 13 deep | 30 | 0 |
| G | Boost pressure, inlet | ISO 11926 | 3/4-16 UNF-2B; 15 deep | 30 | 0 |
| G ₁ | Boost pressure, inlet (only for DA control valve) | ISO 11926 | 3/4-16 UNF-2B; 13 deep | 30 | 0 |
| G ₂ | Boost pressure, outlet (only for boost pump) | ISO 11926 | 3/4-16 UNF-2B; 15 deep | 30 | 0 |
| M _G | Measuring boost pressure G | ISO 11926 | 9/16-18 UNF-2B; 13 deep | 30 | Х |
| M _A , M _B | Measuring pressure A, B | ISO 11926 | 9/16-18 UNF-2B; 13 deep | 420 | Х |

1) Involute spline according to ANSI B92.1a, 30° pressure angle, flat root, side fit, tolerance class 5

2) Thread according to ASME B1.1

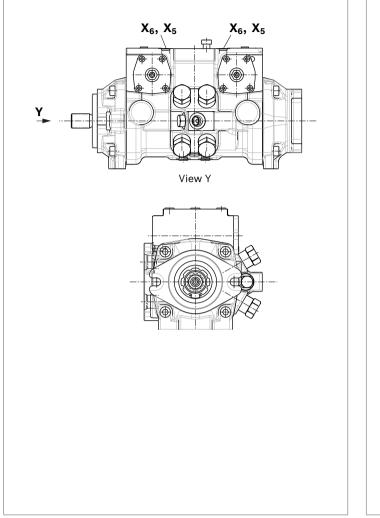
3) The spot face can be deeper than specified in the appropriate standard.

4) For notes on tightening torques, see instruction manual

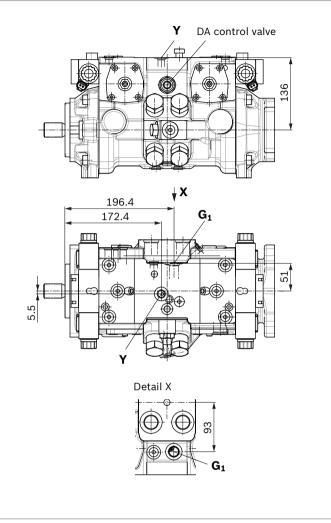
- 5) Momentary pressure spikes may occur depending on the application. Keep this in mind when selecting measuring devices and fittings.
- 6) Optional, see page 21
- 7) O = Must be connected (plugged on delivery)X = Plugged (normal operation)

18 **A22VG series 40** | Axial piston variable double pump Dimensions size 45

▼ HT - Hydraulic control, direct operated



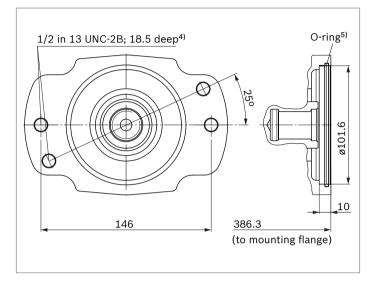
DA control valve



Through drive dimensions

| Flange SAE J744 ¹⁾ Hu | | Hub for | splined shaft ²⁾ | | Availability | Short code | |
|----------------------------------|--------------------------|-------------|-----------------------------|-------------|--------------|------------|------|
| Diameter | attachment ³⁾ | Designation | Diamete | r | Designation | 045 | |
| 101-2 (B) | 0-0 | B2 | 7/8 in | 13T 16/32DP | S4 | • | B2S4 |
| | | | 1 in | 15T 16/32DP | S5 | • | B2S5 |

▼ 101-2



Overview of attachment options

| Through drive | | | Attachment option | Attachment option – additional pumps | | | | |
|------------------|--------------------------|------------|---------------------|--------------------------------------|----------------------------------|--|--|--|
| Flange | Hub for splined shaft | Short code | A10VG NG (shaft) | A10V(S)O/53 NG (shaft) | External gear pump ⁶⁾ | | | |
| 101-2 (B) | 7/8 in | B2S4 | 18 (S) | 28 (S) | Series N, NG20 to 36 | | | |
| | | | | 45 (U) | Series G, NG32 to 50 | | | |
| | 1 in | B2S5 | 28, 45 (S) | 45 (S) | - | | | |
| | | | | 60 (U) | | | | |

Combination pumps

By using combination pumps, it is possible to have independent circuits without the need for splitter gearboxes. When ordering combination pumps, the type designations of the 1st and 2nd pump must be linked by a "+".

Ordering example A22VG045HT100100/40AR + AZPN....

The A22VG variable double pump is permissible without additional supports where the dynamic mass acceleration does not exceed maximum 10 g (= 98.1 m/s²). When mounting another pump on the A22VG, the mounting flange must be rated for the permissible mass torque.

- 1) The through-drive flange is only supplied with the fastening thread corresponding to the ordering code designation.
- According to ANSI B92.1a, 30° pressure angle, flat root, side fit, tolerance class 5
- 3) Mounting drillings pattern viewed on through drive, with control at top

 Bosch Rexroth recommends special versions of the external gear pumps. Please contact us.

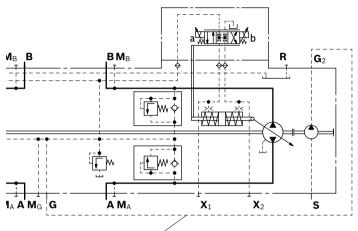
Thread according to ASME B1.1, for notes on tightening torques, see instruction manual

⁵⁾ O-ring included in the scope of delivery

Boost pump

The boost pump continuously supplies a volume of fluid (boost volume) from a reservoir to the low-pressure side of the closed circuit via a check valve to replenish the internal leakage of the variable double pump and consumer. The boost pump is an internal gear pump that is driven directly via the drive shaft. The pressure port G_2 of the boost pump must be externally piped up to port G(or G_1 by the customer for version with DA control valve) (see example circuit diagram below). Suction or pressure filtration is to be provided by the customer.

Schematic



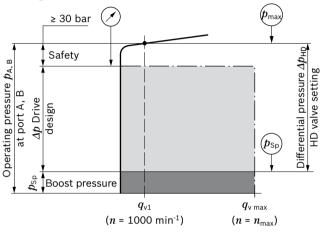
Piping by customer

High-pressure relief valves

The four high-pressure relief valves protect the hydrostatic transmission (pump and motor) from overload. They limit the maximum pressure in the respective high-pressure line and serve simultaneously as boost valves.

High-pressure relief valves are not working valves and are only suitable for pressure spikes or high rates of pressure change.

Setting the valves



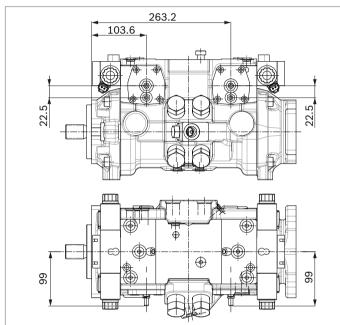
- The valve settings are made at n = 1000 rpm and at V_{g max} (q_{v 1}). There may be deviations in the cracking pressures with other operating parameters.
- The differential pressure setting is preset in the range
 Δp = 250 to 390 bar in increments of 10 bar.
- When ordering, state differential pressure setting in plain text.

| Settings on high-pressure relief valve A and B (Pump 1 and 2) | | | | | |
|--|---------------------------|--|--|--|--|
| Differential pressure setting | $\Delta p_{\rm HD}$ = bar | | | | |
| Cracking pressure of the HD valve (at q_{V1}) ($p_{max} = \Delta p_{HD} + p_{Sp}$) | p_{\max} = bar | | | | |

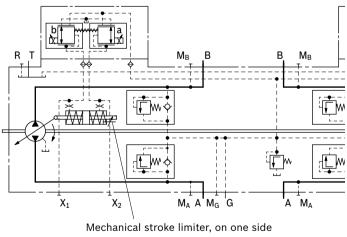
Mechanical stroke limiter

The mechanical stroke limiter is an additional function allowing the maximum displacement of the pump to be steplessly reduced, regardless of the control module used. With one threaded pin per pump, the stroke of the stroking piston and thus the maximum swivel angle per pump is limited on one side.

Dimensions

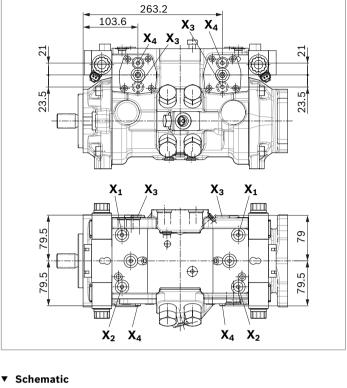


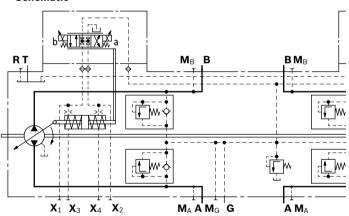
Schematic



Ports X_3 and X_4 for stroking chamber pressure

Dimensions





| Ports | | Standard ¹⁾ | Size [in] ²⁾ | p _{max abs} [bar] ³⁾ | State ⁴⁾ |
|---|---------------------------|------------------------|-------------------------|--|---------------------|
| X ₃ , X ₄ | Stroking chamber pressure | ISO 11926 | 7/16-20 UNF-2B; 12 deep | 30 | Х |

1) The spot face can be deeper than specified in the appropriate standard.

3) Momentary pressure spikes may occur depending on the application. Keep this in mind when selecting measuring devices and fittings.

2) For notes on tightening torques, see instruction manual

4) X = Plugged (in normal operation)

Swivel angle sensor

For the swivel angle indicator, the pump swivel angle is measured by an electric swivel angle sensor.

As an output parameter, the Hall-effect swivel angle sensor delivers a voltage proportional to the swivel angle (see table of output voltages).

Please contact us if the swivel angle sensor is used for control.

| Characteristics | | | | |
|--|--|--|--|--|
| Supply voltage U _b | 10 to 30 V DC | | | |
| Output voltage U _a | 1 V 2.5 V 4 V (V _{g max}) (V _{g 0}) (V _{g max}) | | | |
| Reverse polarity protection | Short circuit-resistant | | | |
| EMC resistance | Details on request | | | |
| Operating temperature range | -40 °C to +115 °C | | | |
| Vibration resistance sinusoidal vibration EN 60068-2-6 | 10 g / 5 to 2000 Hz | | | |
| Shock resistance continuous shock IEC 68-2-29 | 25 g | | | |
| Salt spray resistance DIN 50 021-SS | 96h | | | |
| Type of protection with mounted mating connector | IP67 (DIN/EN 60529) and IP69K (DIN 40050-9) | | | |
| Housing material | Plastic | | | |

Operating

pressure

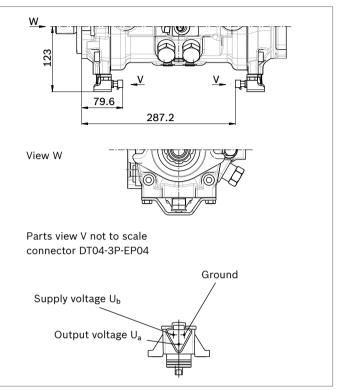
 M_A

MB

MΒ

MA

Dimensions



▼ Mating connector DEUTSCH DT06-3S-EP04

| Consisting of | DT designation | |
|---------------|----------------|--|
| 1 housing | DT06-3S-EP04 | |
| 1 wedge | W3S | |
| 3 sockets | 0462-201-16141 | |

The mating connector is not included in the scope of delivery. This can be supplied by Bosch Rexroth on request (material number R902603524).



Flow

B to A

A to B

A to B

B to A

direction¹⁾

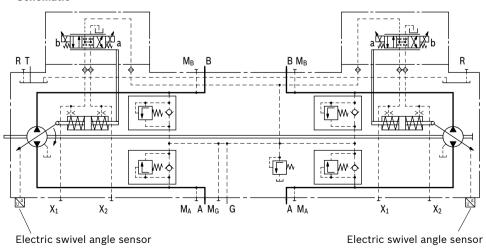
Output voltage

rotation

cw

ccw

▼



Output voltage

> 2.5 V

< 2.5 V

> 2.5 V

< 2.5 V

1) For flow direction, see controls

Connector for solenoids

DEUTSCH DT04-2P-EP04

Molded, 2-pin, without bidirectional suppressor diode. There is the following type of protection with mounted mating connector:

- ▶ IP67 (DIN/EN 60529) and
- ▶ IP69K (DIN 40050-9)

Circuit symbol



▼ Mating connector DEUTSCH DT06-2S-EP04

| Consisting of | DT designation | |
|---------------|----------------|--|
| 1 housing | DT06-2S-EP04 | |
| 1 wedge | W2S | |
| 2 sockets | 0462-201-16141 | |

The mating connector is not included in the scope of delivery. This can be supplied by Bosch Rexroth on request (material number R902601804).

Note

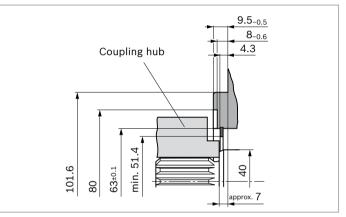
- If necessary, you can change the connector orientation by turning the solenoid housing.
- Refer to the instruction manual for the procedure.

Installation dimensions for coupling assembly

To ensure that rotating components (coupling hub) and fixed components (housing, snap ring) do not come into contact with each other, the installation conditions described here must be observed. This depends on the size and the splined shaft.

SAE splined shaft (spline according to ANSI B92.1a)

The outer diameter of the coupling hub must be smaller than the inner diameter of the snap ring (dimension d_2) in the area near the drive shaft collar (dimension $x_2 - x_3$). Please observe diameter d_5 of the free turning.



Installation instructions

General

During commissioning and operation, the axial piston unit must be filled with hydraulic fluid and air bled. This must also observed following a relatively long standstill as the axial piston unit may drain back to the reservoir via the hydraulic lines.

The leakage in the housing must be directed to the reservoir via the highest drain port **T**.

For combinations of multiple units, the leakage must be drained at each pump. If a shared drain line is used for this purpose, make certain that the respective case pressure is not exceeded. In the event of pressure differences at the drain ports of the units, the shared drain line must be changed so that the minimum permissible case pressure of all connected units is not exceeded in any situation. If this is not possible, separate drain lines must be laid if necessary.

To achieve favorable noise values, decouple all connecting lines using elastic elements and avoid above-reservoir installation.

In all operating conditions, the suction and drain lines must flow into the reservoir below the minimum fluid level. The permissible suction height h_s results from the overall loss of pressure; it must not, however, be higher than $h_{s max}$ = 800 mm. The minimum suction pressure at port **S**

must also not fall below 0.8 bar absolute during operation (cold start 0.5 bar absolute).

When designing the reservoir, ensure adequate space between the suction line and the drain line. This prevents the heated, return flow from being drawn directly back into the suction line.

Installation position

See the following examples 1 to 4.

Further installation positions are possible upon request.

Notes

- If it is not possible to fill the stroking chambers via X₁ to X₄ in the final installation position, this must be done prior to installation.
- To prevent unexpected actuation and damage, the stroking chambers must be air bled via ports X₁, X₂ or X₃, X₄ depending on the installation position.
- For HT control, X₁, X₂ are not present and are replaced by X₅, X₆.
- In certain installation positions, an influence on the control characteristics can be expected. Gravity, dead weight and case pressure can cause minor shifts in control characteristics and changes in response time.

| Key | |
|--------------------|--|
| L | Filling / air bleed |
| R | Air bleed port |
| S | Suction port |
| Т | Drain port |
| SB | Baffle (baffle plate) |
| h _{t min} | Minimum required immersion depth (200 mm) |
| h _{min} | Minimum required distance to reservoir bottom (100 mm) |
| h _{S max} | Maximum permissible suction height (800 mm) |

Above-reservoir installation means that the axial piston unit

Filling

 L_1

 $L_1 + L_2(S) + X_1 + X_2$

 $L_1 + L_2(S)$

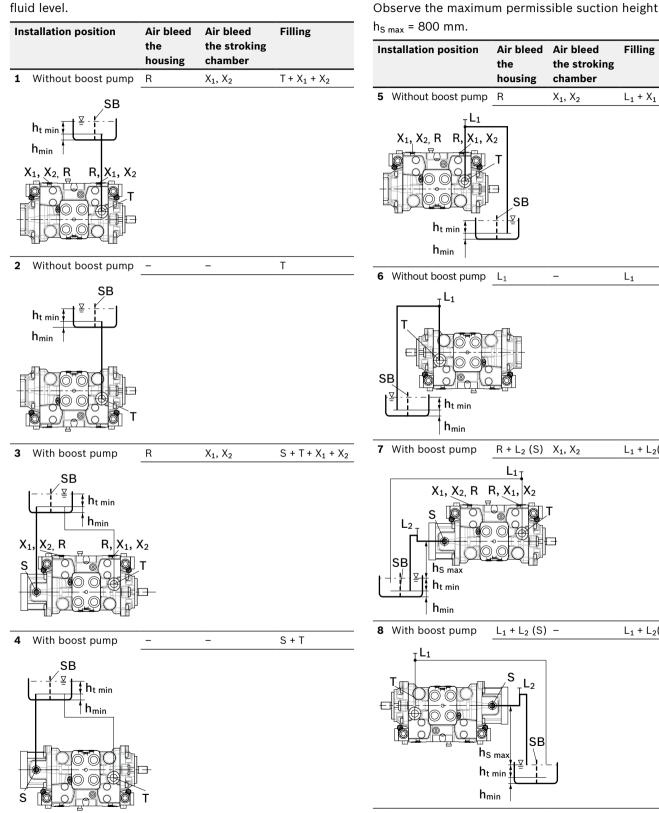
 $L_1 + X_1 + X_2$

is installed above the minimum fluid level of the reservoir.

Above-reservoir installation

Below-reservoir installation (standard)

Below-reservoir installation means that the axial piston unit is installed outside of the reservoir below the minimum fluid level.



For legend and notes, see page 24.

Project planning notes

- ► The pump A22VG is designed to be used in closed circuit.
- The project planning, installation and commissioning of the axial piston unit requires the involvement of qualified skilled person.
- Before using the axial piston unit, please read the corresponding instruction manual completely and thoroughly.
 If necessary, these can be requested from Bosch Rexroth.
- Before finalizing your design, request a binding installation drawing.
- The data and notes contained herein must be adhered to.
- Depending on the operating conditions of the axial piston unit (operating pressure, fluid temperature), the characteristic curve may shift.
- Not all variants of the product are approved for use in safety functions according to ISO 13849. Please consult the responsible contact person at Bosch Rexroth if you require reliability parameters (e.g. MTTF_d) for functional safety.
- Working ports:
 - The ports and fastening threads are designed for the specified maximum pressure. The machine or system manufacturer must ensure that the connecting elements and lines correspond to the specified application conditions (pressure, flow, hydraulic fluid, temperature) with the necessary safety factors.
 - The working ports and function ports can only be used to accommodate hydraulic lines.

Safety instructions

- During and shortly after operation, there is a risk of burns on the axial piston unit and especially on the solenoids. Take appropriate safety measures (e.g. by wearing protective clothing).
- Moving parts in control and regulation systems (e.g. valve spools) may in certain circumstances become stuck in an undefined position due to contamination (e.g. contaminated hydraulic fluid, abrasion or residual dirt from components). As a result, the hydraulic fluid flow or build-up of torque of the axial piston unit will no longer respond correctly to the operator's commands. Even the use of different filter cartridges (external or internal inlet filter) will not rule out a fault but merely minimize the risk. The machine/system manufacturer must test whether remedial measures are needed on the machine for the application concerned in order to set the consumer being driven to a safe position (e.g. safe stop) and if necessary to ensure it is properly implemented.

Bosch Rexroth Corporation

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